APPLIED HEAT

Attempt SIX questions only

All questions carry equal marks

Marks for each part question are shown in brackets

1. A gas in a cylinder is heated at constant volume from a temperature of 387°C to 568°C and then further heated at constant pressure to 827°C.

The change in specific entropy in the first process is 0.182 kJ/kgK.

After the heating processes the gas expands is entropically to 10 times V_1 and a temperature of 17°C.

- (a) Sketch PV and Ts diagrams to show the process (2)
- (b) Calculate EACH of the following:
 - (9) (γ) Cv, CP and γ;
 - (ii) the change is specific entropy in the constant pressure process; (2)
 - (iii) the specific work in the isentropic process. (3)
- 2. In an air standard dual combustion cycle, the initial volume compression ratio is 13/1.

The maximum and minimum temperatures are 1500 K and 300 K, respectively.

The maximum and minimum pressures are respectively 67 bar and 1.13 bar.

- (a) Sketch the processes on p-V and T-s diagrams. (2)
- (b) Calculate EACH of the following:
 - (i) the temperature at all points; (2)
 - (ii) the thermal efficiency,
 - (iii) the mean effective pressure. (2)

Note: For air y = 1.4 and Cp = 1.005kJ/kgK Cv = 0.718kJ/kgK

| 3. | A single acting single stage air compressor runs at 300 rpm and has a clearance volume equal to 7% of the swept volume. | |
|----|--|------|
| | The compressor delivers 50 m³ of air per hour measured at 1.013 bar and 0°C. | |
| | The suction pressure and temperature are 0.98 bar and 23°C (T_1 and T_4) respectively, and the discharge pressure and temperature are 6.6 bar and 170°C (T_2 and T_3), respectively. | |
| | (a) Sketch the cycle on a pressure-volume diagram. | (2) |
| | (b) Calculate EACH of the following: | |
| | (i) the volume induced per cycle; | (5) |
| | (ii) the swept volume; | (4) |
| | (iii) the index of compression; | (3) |
| | (iv) the indicated power. | (2) |
| | Note: For Cp= 1.005kJ/kgK Cv=0.718kJ/kgK | |
| | | |
| 4. | A boiler generates 5000 kg of steam per hour at 15 bar. The steam temperature is 325°C and the feedwater temperature is 48°C. | |
| | If the fuel supplied has a calorific value of 45.5 MJ/kg and is supplied at a rate of 0.11 kg/s. On exiting the boiler, the steam is further heated to 450°C before being throttled to 10 bar. | |
| | (a) Calculate the boiler efficiency. | (10) |
| | (b) Find the supplied heat required in the superheater. | (3) |
| | (c) Draw the process on a pressure enthalpy diagram. | (3) |
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| 5. | A vapour compression refrigeration system operates between the pressures of 4.625 bar and 8.053 bar. | |
|----|--|------|
| | R717 refrigerant enters the compressor dry saturated and is isentropically compressed. Upon leaving the condenser it is undercooled by 6 K. The mass flow rate is 0.8 kg/s | |
| | (a) Sketch the cycle on pressure-specific enthalpy and Temperature-specific entropy diagrams. | (2) |
| | (b) Calculate EACH of the following: | |
| | (i) The dryness fraction of the refrigerant entering the evaporator; | (3) |
| | (ii) The temperature leaving the compressor; | (5) |
| | (iii) The compressor power. | (4) |
| | | (2) |
| | (c) Explain the application of intermediate liquid cooling | |
| | | |
| 6. | An LNG carrier has five spherical tanks each of diameter 36 m. They contain liquefied gas at a temperature of-163°C. The tanks are insulated with 60 cm thickness of material of fibreglass with a thermal conductivity of 0.04 W/mK. | |
| | The outside surface heat transfer coefficient is 8 W/m²K and the outside air temperature is 28°C. | |
| | (a) Calculate the total rate of heat gain. | (7) |
| | (b) Find the external surface temperature. | (4) |
| | (b) Find the external surface temperated each day. | (5) |
| | (c) Calculate the total percentage mass evaporated each day. | |
| | Note: $\rho = 1000 kg/m^3$ Latent heat of evaporation is 515kJ/kg | |
| | note: P | |
| | 42% excess air | |
| 7. | Butane (C ₄ H ₁₀) is burned with 12% excess air. | (14) |
| | (a) Analyse the percentage of dry products by volume. | (2) |
| | The HCV | (2) |
| | (b) Define the new contraction of the contraction o | |
| | | |

8. An impulse turbine has a nozzle at the entrance angle is 16°, the blade velocity is 360 m/s, and the blades are symmetrical and have an angle of 34°.

If the blade velocity coefficient is 0.94 and the mass flowrate of the steam is 1.67 kg/s.

- (a) Calculate the nozzle exit velocity if the change in enthalpy across the nozzle is 490 kJ/kg. (2)
- (b) Sketch a blade velocity diagram labelling all significant angles and velocities. (3)
- (c) Using Fig Q8, calculate the diagram power. (11)

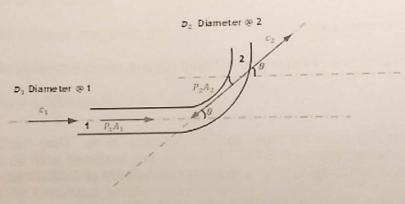


Fig Q8

9. A bend of constant cross-sectional area which turns through 75 degrees is fitted in a horizontal section of a 600 mm diameter fresh water-cooling system.

The cooling system pressure is 3 bar and the flowrate is 0.85 m3/s. The pressure loss due to each bend is negligible.

Calculate EACH of the following:

- (a) the net force acting on the axis ox; (6)
- (b) the net force acting on the axis oy; (6)
- (c) the magnitude of the resultant force acting on the bend; (2)
- (d) the direction of the resultant force. (2)